

FEEDER DEVICE IN A TIMBER HARVESTER

The present invention relates to a feeder device in a timber harvester, which feeder device includes a 3-row roller chain  
5 arranged to be rotated around a drive sprocket, a turnover member and rolling guides, which extend over a great length of the adhesion side between the drive sprocket and the turnover member, and in which roller chain there are rows of links staggered relative to each other by transverse pins, comprising  
10 a middle row of links and outer rows of links, each row of links including rollers rolling in the corresponding rolling guides and set in bearings in the transverse pins, and in which the drive sprocket is arranged to drive by its teeth the middle row of links of the roller chain through its rollers. In this  
15 case, the term 'timber harvester' must be understood quite broadly as encompassing very different kinds of devices for handling fibre and log timber.

A timber harvester feeder device according to the preamble is  
20 known from the applicant's previous publications WO 85/05589 and WO 99/25526. A timber harvester usually has two feed tracks jointed oppositely to each other, stripping blades, and a cutting device. The frame of the feed track is installed on articulated arms. The roller chain is fitted around a drive  
25 sprocket, driving the middle row of links, and a turnover member, generally a pair of idler wheels. The dimensions of the roller chain are adapted to the dimension of the toothing of the drive sprocket, which has limited the diameter of the rollers.

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A track feed has obvious advantages over drive rollers. The first publication deals with a situation, in which rolling bases and roller chains are used. This substantially improves the durability of the track feed, as the solution allows the  
35 loading to be transferred directly to the sliding base, thus avoiding loading peaks in the track. The latter publication

discloses a curved rolling base, so that the track-feed device imitates a large drive roller, the length of the grip with the timber being much longer than in any real drive roller, the diameter of which remains much smaller for practical reasons.

5 Due to the curved rolling base, the mechanism is simplified, as one joint of the feed device can be eliminated. A crawler track is a special roller chain, in which the rollers and pins wear in use. In order to reduce friction, the rollers are equipped with bushings. Correspondingly, the rolling bases wear in use

10 and, like the crawler track, are preferably made to be replaceable. In known rolling bases, grooves must be machined between the rollers, to prevent the links from coming in contact with the base.

15 The present invention is intended to create an improved feeder device for a timber harvester, which will have a long operating life and excellent reliability. The feeder device is an essential component in a timber harvester and has a significant effect on the total weight of the entire machine. The invention

20 is intended to reduce the size of and weight of the track feeder device, without impairing its durability. These objectives are achieved by means of the features stated in Claim 1. The invention is largely based on the observation that the rolling base mainly wears only at the sides, but that the size

25 of the outer rows of links is determined by the middle row of links, in which the rollers are adapted to the driving sprocket. Wear is thus generally limited to the side rows of the crawler track. The row in the middle of the track generally does not wear, even though it is subjected to the stress of the

30 drive sprocket driving the track. Therefore the outer rows of links can, in fact, be equipped with larger rollers, provided that the rolling base is also adapted to rollers of different sizes.

35 In the following, the invention is examined with the aid of one embodiment, which is shown in the accompanying drawings:

Figure 1 shows a timber harvester, equipped with track-feed devices

Figure 2 shows a side cross-section of the feed device

Figure 3 shows a top view of the feed device of Figure 2, with  
5 the crawler track partly removed

Figure 4 shows a cross-section of the feed device, through the rolling base.

Figure 1 shows one timber harvester 10, which includes a frame  
10 11, a suspension device 12, two feed tracks 14 jointed oppositely to each other, stripping blades 15, and a cutting device 16 (a saw). The feed tracks 14 and the stripping blades 15 are operated with the aid of hydraulic cylinders (not shown) and they press the tree being processed into their throat. The feed  
15 tracks 14 pull the tree through the throat.

The frame of the feed track 14 is installed on articulated arms 13. The feed track 14 is a 3-row roller chain fitted around a drive sprocket, a turnover member, and rolling guides, which  
20 extend over a great length of the gripping side, between the drive sprocket and the turnover member.

The weight of the timber harvester can be reduced by making smaller crawler-track feeder devices. The pressure, on the  
25 surface of the timber, of a crawler track rolling even on a curved base is not too great, even though the contact surface between the track and the timber is shorter than in a feeder device equipped with a straight rolling base. The reduction in weight is therefore limited only by the surface pressure be-  
30 tween the crawler track and its base and subsequently by the wear of their components that roll mutually relative to each other.

In the feeder device according to the invention, the crawler  
35 track is a 3-row track. Though the chain can be a traditional 3-row roller chain, in which the links are next to each other,

they are preferably staggered relative to each other (WO 85/05589), in order to equalize the loading. The drive sprocket is located at the middle row, so that the dimensions of the construction of this row is based on the drive event. Both the  
5 outer rows of the crawler track are, in turn, equipped with rollers of the greater possible diameter. This creates a 3-row crawler track, in which the diameter of the rollers of the inner row is a maximum of 80 % of their spacing and usually less. In turn, the rollers of the outer rows have a diameter  
10 that is at least 85 % of the spacing of the elements of the crawler track. In practice, the roller diameter of the inner row of the roller chains is about 70 % of the spacing. In the outer rows, a diameter that is 85 - 95 % of the spacing can be preferably used. It can be stated that in general the outer  
15 rollers 33.1 have a diameter 10 - 25% greater than that of the middle rollers 34.1. Preferably, at least the outer rollers 33.1 are equipped with bushings 33.3.

If the diameter of the outer row increases, the surface pressure against the rolling base will decrease, the velocity  
20 between the roller and the internal bushing will decrease, and simultaneously the mutual rolling distance between them will decrease. This all increases the life of the crawler track. But now the rolling base (except for the middle row) can also be  
25 advantageously made in the form of plates, without separate rails. As the rollers of the outer rows are large, the outer surface of the roller is pushed outside of the outer surface of the side plates. No space is then required on the rolling base for the side plates of the outer rows. A rolling base of this  
30 kind can be manufactured effectively using large blades, even though more material must be removed than in a base equipped with grooves.

Figure 2 shows one construction of feeder device 14 (cross-  
35 section B - B, Figure 3). In this case, the feeder device, which is seen in part cross-section from the side, is equipped

with a curved rolling base 22. The tree being processed is marked with the reference number 1. The crawler track is marked generally with the reference number 17 and is driven by a drive sprocket 24. At the opposite end, it travels around two idlers 5 25, which are, however, narrow in the area of the outer rollers and preferably have truncated teeth. The idler can, however, also be smooth, or be replaced entirely with a rolling guide. The teeth 24.1 of the drive sprocket 24 transmit power to the middle row of links of the crawler track 17. The need for these 10 teeth 24.1 to fit between the opposing rollers limits the size of the rollers relative to their spacing. This limitation does not apply to the outer rows of links, and in them the diameter of the rollers can approach the dimension of their spacing.

15 Figure 3 shows a top view of the feeder device 14, with the crawler track 17 nearly entirely removed and partly cut open. The figure shows the rolling base 22, the drive sprocket 24, the hydraulic motor 19 that rotates it, the suspension arm 13.2 of the feeder device, the suspension joint 19, and the idlers 20 25. The crawler track 17 fills the space between the side plates 14.2 of the frame with a small tolerance. Preferably, the crawler track 17 is staggered according to the figure. The pin 17.4 locked to the side plates 17.3 of the crawler track 17 secures the links 33.2 and 34.2 in such a way that the links 25 34.2 of the middle row 17.2 are 'in the same phase' as side plates 33.2, while the links 33.2 of the outer rows of links 17.1 are staggered relative to them. The detachable rolling base 22 is substantially narrower than the crawler track 17, so that at least the side plates 17.3 have plenty of space to 30 move.

Figure 4 shows a cross-section of the feeder device, at the point A - A in Figure 2. The components 33.2 and 34.2 show the links of a 3-row chain. Component 17.3 is the side plate on 35 both sides of the crawler track 17, i.e. the pin 17.4 is attached to them at both ends. In this case, the links 33.3 and

34.3 are formed in a known manner from U-shaped pieces, with grip studs welded to the web. The links can also be made from separate side plates, with a web plate welded across their ends. The gripping members can be a stud welded onto the web  
5 plate, or a plate-like piece.

As will be noticed, the roller 33.1 in the links 33.2 is larger than the roller 34.1 in the links 34.2. These correspond to the rolling bands 22.1 and 22.2 of the rolling base. The rolling  
10 base 22 is otherwise uniform, but is thicker under the middle row, to raise the rolling band 22.2 to correspond to the smaller roller 34.1. The rolling base can also be partial and divided (not shown). In this case, bolts 22.3 are used to make it detachable, but a welded joint can also be used.

15 The rolling guides are preferably curved, with their curvature corresponding to a radius of about 1 metre, usually 0,8 - 1,3 m. The durability, mechanical operation, and adhesion properties will then be optimal.

20 Carbon tempering increases the low surface carbon content of steel to the level 0,65 - 0,9 %, in order to improve its hardenability and achieve a high surface hardness. In this case, the thickness of the carbonization layer is preferably in  
25 the range of 1 - 2,5 mm.

As an alternative to carbon tempering, it is possible to use either tool steel, or induction tempering in a base material with a high carbon content.

30 It is obvious that the invention can be varied within broad limits within the scope of the accompanying Claims. The basic construction of the timber harvester can even deviate greatly from the example shown. Reference is made above to a 3-row  
35 roller chain, meaning three bands. In principle, each band could have two rows of rollers of the same size. Preferably,

the term timber harvester refers to a single-grab harvester, in which there are two opposing feeder devices, the basic construction of which is shown in the said PCT publication WO 99/25526.